



## Electroacoustic absorbers

low frequency absorption by hybrid  
sensor/shunt-based impedance control

Dr. Hervé Lissek, Dr. Sami Karkar, Etienne Rivet (EPFL)

### WARNING:

all sounds in this slideshow should be listened to with a subwoofer or (good) headphones.

# Outline



- Context: room modes in the low-frequency range
- Presentation of the Electroacoustic Absorber prototype
- Design Methodology for room modal damping
- Experimental assessment: modal damping in rooms



# ELECTROACOUSTIC ABSORBERS PRESENTATION

# Electroacoustic absorbers

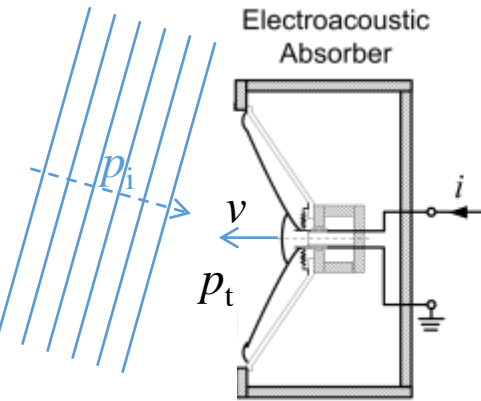


- Development of an active sound absorption solution with  $\alpha > 0.83$  along the range [20 - 200Hz]
- Based on actuated membranes (loudspeakers) used as a membrane absorbers
- Evaluation in reverberant conditions, focusing on low-frequency sound «equalization»



H. Lissek, R. Boulandet, and R. Fleury, "Electroacoustic absorbers: bridging the gap between shunt loudspeakers and active sound absorption", J. Acoust. Soc. Am., 129(5), 2968-2978, (2011).

# Electroacoustic absorbers



$$\begin{cases} S_d p_t = Z_m(\omega) v + B l i \\ \text{with } Z_m(\omega) = R_{ms} + j\omega M_{ms} + \frac{1}{C_{ms}} + \frac{\rho c^2 S_d^2}{V_b} \end{cases}$$

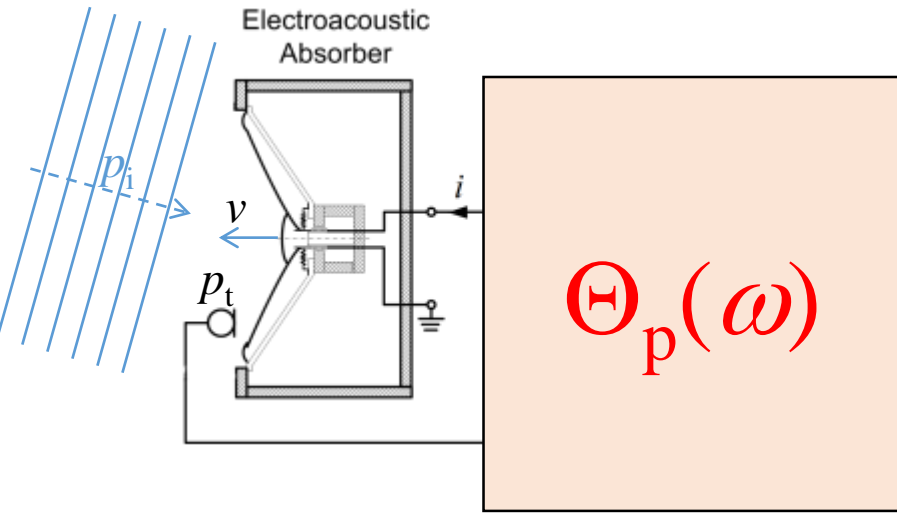
Parameter	Description	Value	Unit
$M_{ms}$	Moving mass	14.7	g
$R_{ms}$	Mechanical resistance	1.31	N.s.m <sup>-1</sup>
$C_{ms}$	Mechanical compliance	242.3	mm.N <sup>-1</sup>
$S_d$	Membrane surface	151	cm <sup>2</sup>
$Bl$	Force factor	6.85	N.A <sup>-1</sup>
$V_b$	Cabinet volume	10	dm <sup>3</sup>
$\rho$	Air mass density	1.2	kg/m <sup>3</sup>
$c$	Sound celerity in air	344	m.s <sup>-1</sup>

# Electroacoustic absorbers



Objective:

force the membrane to behave as a given acoustic impedance  $Z_{at}$

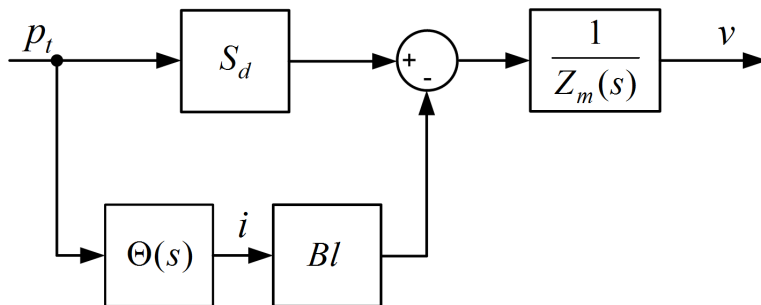


$$\begin{cases} S_d p_t = Z_m(\omega) v + Bl \Theta_p(\omega) p_t \\ Z_a(\omega) = \frac{p_t}{v} = \frac{Z_m}{S_d - Bl \Theta_p(\omega)} \end{cases}$$

Effective acoustic impedance of the membrane

The **desired acoustic impedance**  $Z_{at}$  can be assigned by identifying the controller transfer function:

$$\Theta_p(\omega) = \frac{S_d Z_{at} - Z_m(\omega)}{Bl Z_{at}}$$



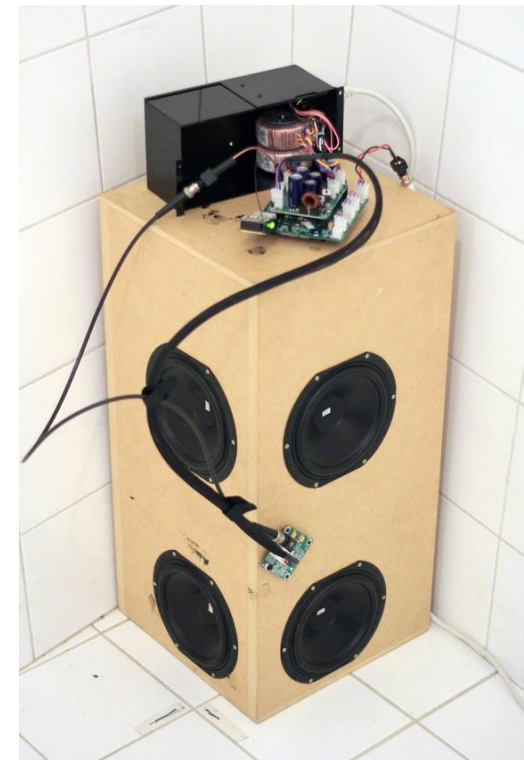
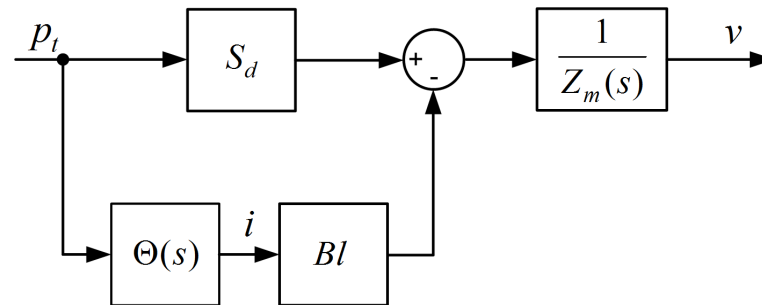
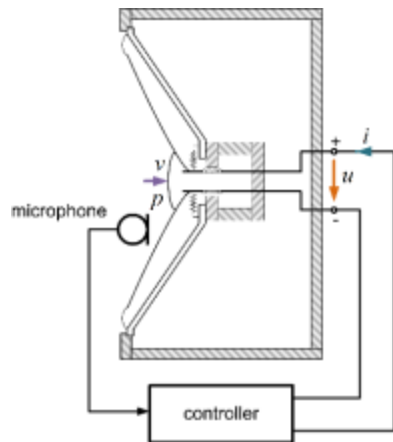


# EXPERIMENTAL VALIDATION OF ROOM MODES DAMPING

# Experimental assessment



- 4x4 electroacoustic absorbers prototypes (total surface =  $16 \times 151 \text{ cm}^2 = 0.24 \text{ m}^2$ )
- In a reverberant chamber of  $226.9 \text{ m}^3$   
➔ evaluation of room modal damping





# Experimental assessment



## 1. Frequency response without and with absorbers

- ➔ identify individual room modes
- ➔ assess damping performance on peaks and dips amplitudes

## 2. Modal decay times without and with absorbers

- ➔ assess damping performance in the time domain

## 3. Recording of music rendering, without and with absorbers

- ➔ listen to the effect on music rendering

## 4. Recording of kick drum, without and with absorbers

- ➔ listen to the effect on acoustic music playing

# Experimental setup



## Hardware

### • Recorders/analyzers

- B&K Pulse (frequency responses)
  - frequency resolution: 31.5 mHz
- M-Audio M-Track 8 soundcard (recordings)

### • Microphones

PCB 130D20  
(frequency responses)



Beyerdynamic M101 N  
(recordings)



### • Sources Subwoofer



Kick drum  
(Pearl Export)



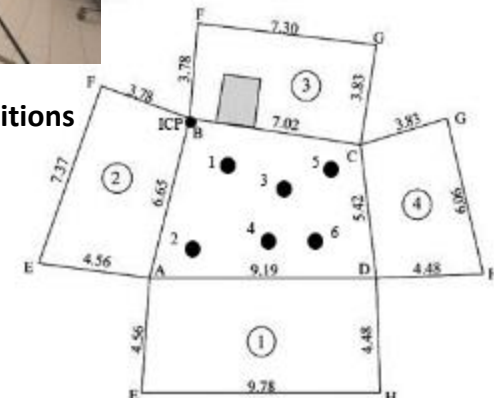
Facility (reverberant chamber,  
 $V=215.6 \text{ m}^3$ ,  $S=226.9 \text{ m}^2$ )



4 electroacoustic absorbers  
at the 4 room corners

Supplementary  
panel absorber

7 microphone positions

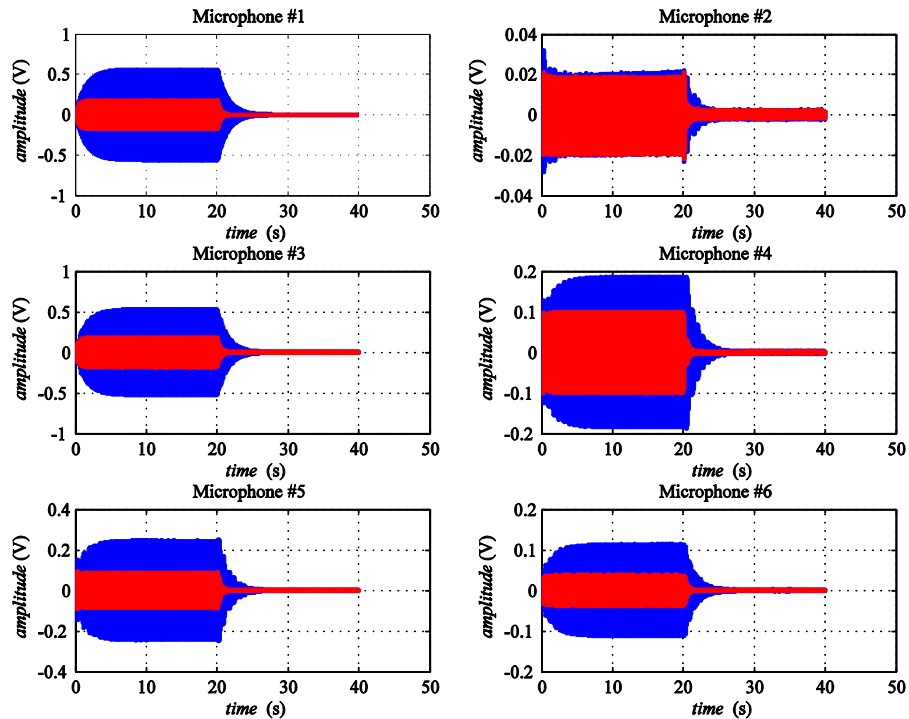


Id	height m	Distance to corner A m	Distance to corner B m	Distance to corner C m	Distance to corner D m
ICP	0.83	-	0	-	-
Micro 1	1.83	4.89	3.35	-	-
Micro 2	2.79	2.07	5.55	-	-
Micro 3	1.81	6.40	4.90	-	-
Micro 4	2.20	-	4.17	3.31	-
Micro 5	1.22	-	-	1.62	5.00
Micro 6	1.49	-	-	4.00	2.84

# 2. Modal decay time

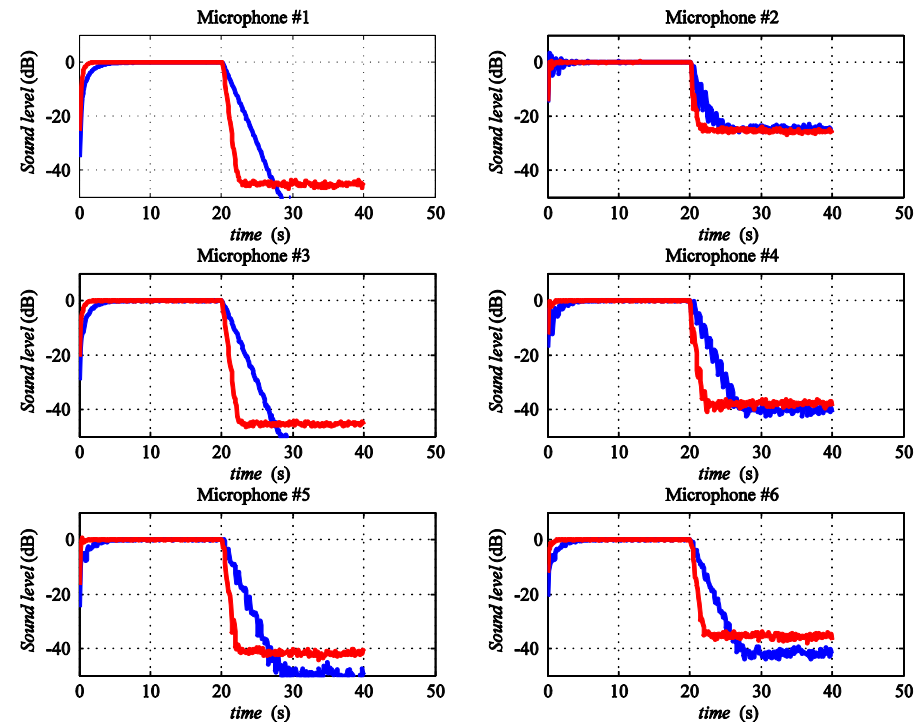
## Mode 8 – waveforms

(hardwalls : 52.78 Hz - absorbers : 52.72 Hz)

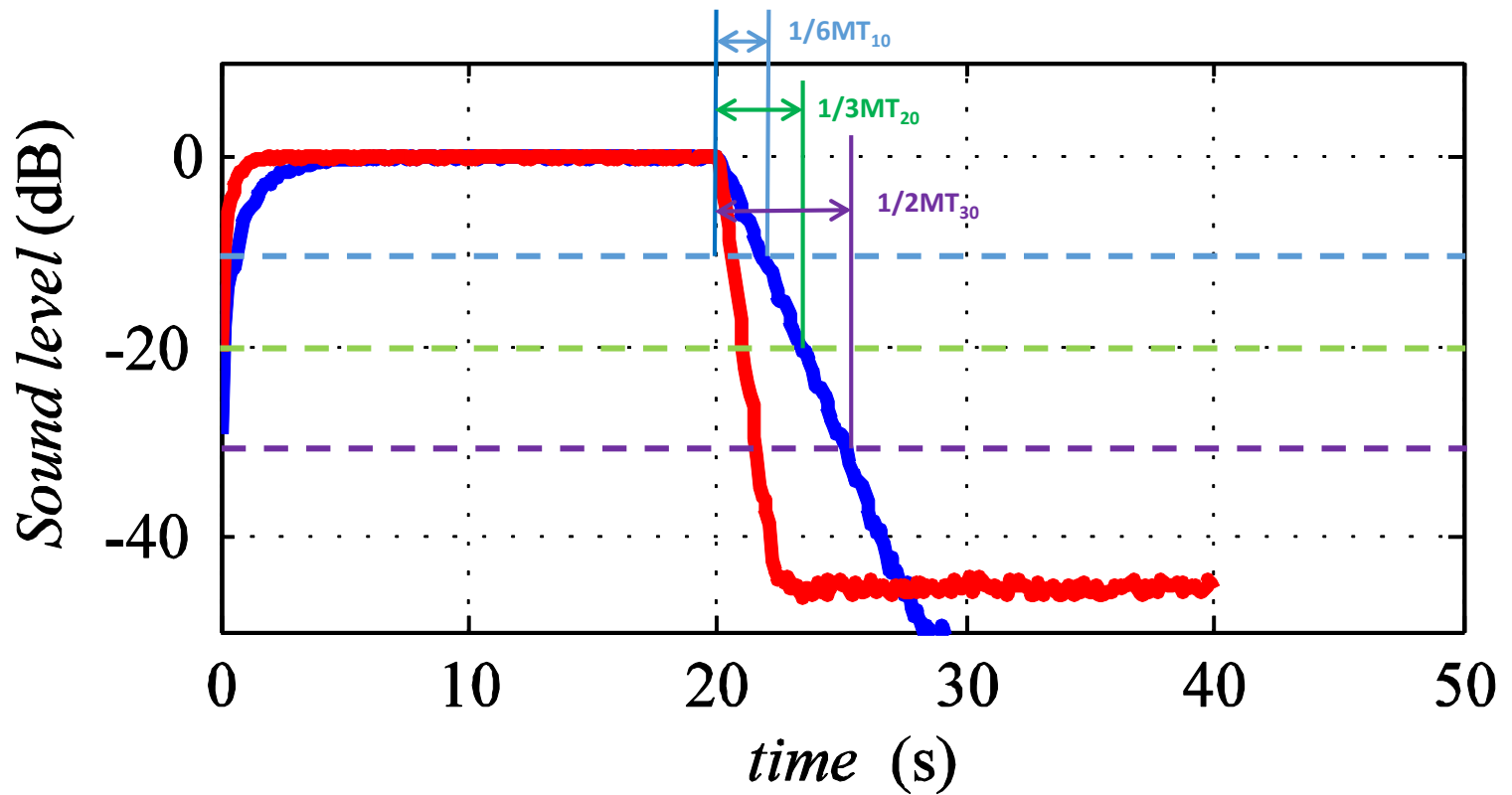


## Mode 8 - echograms

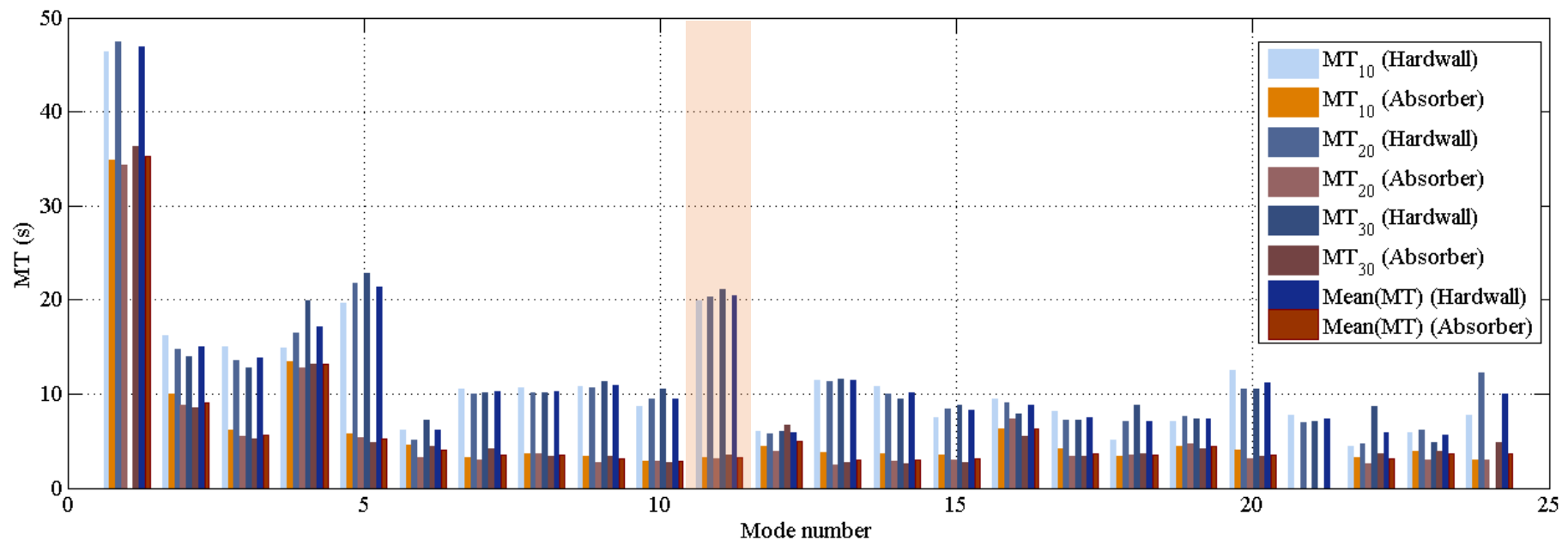
(hardwalls : 52.78 Hz - absorbers : 52.72 Hz)



## 2. Modal decay time



## 2. Modal decay time



# Conclusions



- 4 electroacoustic absorbers prototypes achieve efficient room mode damping in the reverberant chamber
  - Max damping: 12.2 dB @ 58 Hz,
  - Global damping: 8 dB over [20 – 100 Hz],
  - Max modal decay time reduction: 85% @ 58 Hz  
(from 20 s down to 3 s),

for a total absorber surface representing only 0.1% of the whole room walls surface

# Conclusions



Most recent results (August 2015)

